

Port-Hamiltonian Macroscopic Modelling based on the Homogenisation Method: case of an acoustic pipe with a porous wall

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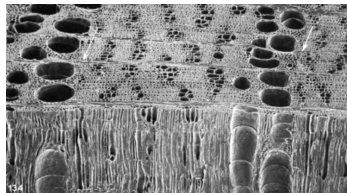
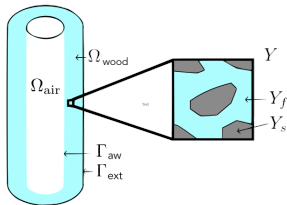


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Motivation

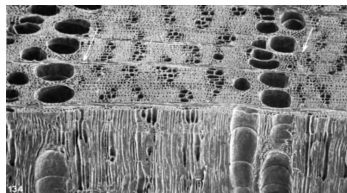
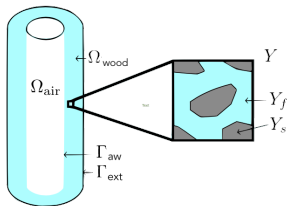
In some woodwind musical instruments, **acoustic waves** are subjected to **volume dissipation inside the wall**



zoom $\times 80$ – *Ulmus procera* Salisb.
(extracted from Butterfiled et al., 1972)

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Audible but slight effect

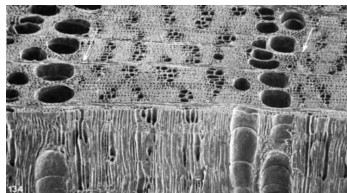
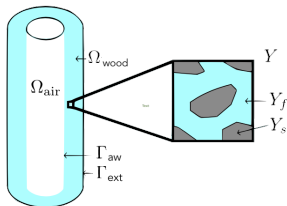
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Dependence on the pore geometry

→ complex model

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Objective

Derive a model that is:

(i) Macroscopic

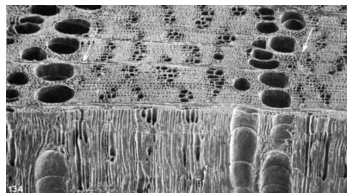
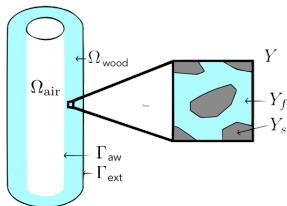
→ Porosity effect experienced at the wall scale
by acoustic waves inside the pipe

(ii) Guaranteed passive

→ Port Hamiltonian Framework

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Work in the framework of PHS (\equiv Port-Hamiltonian System)

1. METHOD : modelling acoustic waves in the porous wall (*+coupling with pipe*)

microscopic
PHS

scale
separation
→

two-scale
PHS

homogenised
problem
→

macroscopic
PHS

2. ILLUSTRATION on a straight acoustic pipe with a thick porous wall

Outline

- A. Motivation
- B. Fluid equations inside the porous wall
- C. Scale separation method: periodic wall homogenisation
- D. Macroscopic loss operators in the Laplace domain and application
- E. Application to an acoustic pipe with a porous wall
- F. Conclusion

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Porous wall: Hypotheses

Domains

Porous wall = Solid part \cup Fluid part

Fluid domain $\Omega_f \subset \mathbb{R}^d$: bounded connected open set

Interface $\Gamma_f \subset \partial\Omega_f$: regular boundary between Ω_f and the solid part

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Fluid with small fluctuations

Field quantity	mass density [kg.m ⁻³]	particle velocity [m.s ⁻¹]	pressure [Pa]	temperature [K]
total	ρ_{tot}	\mathbf{v}_{tot}	P_{tot}	T_{tot}
at rest	$\rho_0 = 1.2$	$\mathbf{v}_0 = \mathbf{0}$	$P_0 = 101.3$	$T_0 = 293.15$ ($\approx 20^\circ\text{C}$)
fluctuation	$\rho = \rho_{\text{tot}} - \rho_0$	$\mathbf{v} = \mathbf{v}_{\text{tot}} - \mathbf{0}$	$P = P_{\text{tot}} - P_0$	$T = T_{\text{tot}} - T_0$

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Material characteristics

Solid part = isothermal ideally rigid wood $T_0 = 293.15 \text{ K} \approx 20^\circ\text{C}$.

Fluid part = air = perfect gas with law
gas constant

specific heat capacity at constant pressure

heat capacity ratio

thermal conductivity

shear viscosity

bulk viscosity

$$P_{\text{tot}} = R_0 \rho_{\text{tot}} T_{\text{tot}}$$

$$R_0 = 288 \text{ J.kg}^{-1}.\text{K}^{-1}$$

$$C_p = \frac{\gamma R_0}{\gamma - 1} \approx 1004 \text{ J.kg}^{-1}.\text{K}^{-1}$$

$$\gamma = 1.402.$$

$$\kappa \approx 2.57 \times 10^{-2} \text{ J.m}^{-1}.\text{s}^{-1}.\text{K}^{-1},$$

$$\mu = 1.81 \times 10^{-5} \text{ kg.m}^{-1}.\text{s}^{-1}$$

$$\zeta = 1.3 \times 10^{-5} \text{ kg.m}^{-1}.\text{s}^{-1}$$

Thermo-visco-acoustic equations

Under these assumptions, the governing equations can be approximated by the

Linearised Navier-Stokes equations

(see e.g. [RUY16, BP13])

Mass conservation in Ω_f

$$\partial_t \rho + \rho_0 \operatorname{div} \mathbf{v} = 0, \quad (1a)$$

Momentum conservation in Ω_f

$$\rho_0 \partial_t \mathbf{v} = -\nabla P + \mu \Delta \mathbf{v} + \left(\zeta + \frac{\mu}{3} \right) \nabla(\operatorname{div} \mathbf{v}), \quad (1b)$$

Thermal conduction in Ω_f

$$\rho_0 C_p \partial_t T = \kappa \Delta T + \partial_t P, \quad (1c)$$

Gas state equation in Ω_f

$$P/P_0 = \rho/\rho_0 + T/T_0, \quad (1d)$$

No-slip condition on Γ_f :

$$\mathbf{v} = 0, \quad (1e)$$

Thermal contact on Γ_f :

$$T = 0. \quad (1f)$$

Port-Hamiltonian formulation (1/2)

Using the gas state equation (1d) to substitute ρ , equations (1a-1c) rewrite:

Conservation and conduction equations on effort variables P, \mathbf{v}, T

$$\underbrace{\begin{pmatrix} \frac{1}{P_0} & 0 & -\frac{1}{T_0} \\ 0 & \rho_0 & 0 \\ -\frac{1}{T_0} & 0 & \frac{\rho_0 C_p}{T_0} \end{pmatrix}}_{\mathcal{M}} \underbrace{\begin{pmatrix} \partial_t P \\ \partial_t \mathbf{v} \\ \partial_t T \end{pmatrix}}_{\partial_t \mathbf{e}} = \underbrace{\begin{pmatrix} 0 & -\operatorname{div} & 0 \\ -\nabla & \mathcal{A} & 0 \\ 0 & 0 & \frac{\kappa}{T_0} \Delta \end{pmatrix}}_S \underbrace{\begin{pmatrix} P \\ \mathbf{v} \\ T \end{pmatrix}}_e, \quad (2)$$

with $\begin{cases} \mathcal{M} = \mathcal{M}^\top \succ 0 & (\text{positive as } \rho_0 C_p / (P_0 T_0) = \gamma / (\gamma - 1) > 1) \\ \mathcal{A} = \mu \Delta + (\zeta + \frac{\mu}{3}) \nabla(\operatorname{div}(\cdot)) & : \text{negative self-adjoint operator} \end{cases}$

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... structured by operator $\mathcal{S} = \mathcal{J} - \mathcal{R}$ with

$$\underbrace{\mathcal{J} = -\mathcal{J}^*}_{\text{skew-symmetric}} = \begin{pmatrix} 0 & -\text{div} & 0 \\ -\nabla & 0 & 0 \\ 0 & 0 & 0 \end{pmatrix} \text{ and } \underbrace{\mathcal{R} = \mathcal{R}^*}_{\text{symmetric}} = \begin{pmatrix} 0 & 0 & 0 \\ 0 & -\mathcal{A} & 0 \\ 0 & 0 & \frac{\kappa}{T_0} (-\Delta) \end{pmatrix} \underbrace{\succeq 0}_{\text{pos.}}$$

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This defines a **linear dissipative port-Hamiltonian system**

(formulated with co-energy (\equiv effort) variables \mathbf{e} , see [VdSJ⁺14])

Energy: $E = \frac{1}{2} \langle \mathbf{e}, \mathcal{M} \mathbf{e} \rangle_{\Omega_f} = \frac{1}{2} \int_{\Omega_f} \mathbf{e}^T \mathcal{M} \mathbf{e} \, d\Omega \geq 0$

Power balance: $\dot{E} = \langle \mathbf{e}, \mathcal{M} \partial_t \mathbf{e} \rangle_{\Omega_f} \stackrel{(2)}{=} \langle \mathbf{e}, \mathcal{S} \mathbf{e} \rangle_{\Omega_f} = - \underbrace{\langle \mathbf{e}, \mathcal{R} \mathbf{e} \rangle_{\Omega_f}}_{\text{dissipated power}} \leq 0$ (if no flow or effort on $\partial\Omega_f$)

Port-Hamiltonian formulation (2/2)

Consider the energy variable α and Hamiltonian $H(\alpha)$ s.t. $\delta_\alpha H = \mathbf{e}$

$$\alpha := \mathcal{M} \mathbf{e} = \begin{pmatrix} \frac{P}{P_0} - \frac{T}{T_0} = \frac{\rho}{\rho_0} \\ \rho_0 \mathbf{v} \\ \rho_0 C_P \frac{T}{T_0} - \frac{P}{T_0} \end{pmatrix} \begin{array}{l} \text{relative mass density,} \\ \text{momentum density,} \\ \equiv \text{entropy density,} \end{array} \quad (3a)$$

$$H(\alpha) := \frac{1}{2} \langle \mathcal{M}^{-1} \alpha, \alpha \rangle_{\Omega_f} = \frac{1}{2} \langle \mathbf{e}, \mathcal{M} \mathbf{e} \rangle_{\Omega_f} = E \quad (3b)$$

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$\underbrace{\hspace{15em}}_{(3a) \partial_t \alpha} \qquad \underbrace{\hspace{15em}}_{(3a) \mathcal{M}^{-1} \alpha \stackrel{(3b)}{=} \delta_\alpha H(\alpha)}$

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$\underbrace{\hspace{15em}}_{\equiv \partial_t \alpha} \qquad \underbrace{\hspace{15em}}_{\equiv \mathcal{M}^{-1} \alpha \equiv \delta_\alpha H(\alpha)}$

This yields the standard state-space Port-Hamiltonian System

microscopic
PHS in Ω_f

$$\partial_t \alpha = (\mathcal{J} - \mathcal{R}) \delta_\alpha H(\alpha), \quad (4)$$

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Power balance: $\frac{d}{dt} H(\alpha(t)) = \langle \delta_\alpha H, \partial_t \alpha \rangle_{\Omega_f} = \langle \delta_\alpha H, (\mathcal{J} - \mathcal{R}) \delta_\alpha H \rangle_{\Omega_f}$

$$= - \underbrace{\langle \delta_\alpha H, \mathcal{R} \delta_\alpha H \rangle_{\Omega_f}}_{\text{dissipated power} \geq 0} \leq 0 \quad (\text{if no flow or effort on } \partial\Omega_f)$$

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B. Fluid equations inside the porous wall

C. Scale separation method: periodic wall homogenisation

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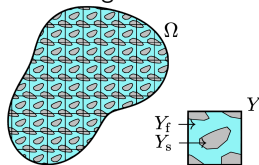
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Homogeneisation problem

1. Introduce a **scaling coefficient** $\varepsilon = L_p/L_m \ll 1$ to separate the **microscopic pore length** L_p at which **dissipation** takes place from the *macroscopic* length L_m at which *acoustic propagation* takes place

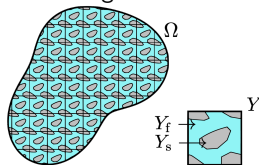
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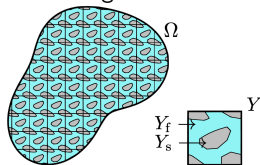
3. Distinguish scales using

Macroscopic coordinate $\mathbf{x} \in \Omega$,

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4. In the governing equations, replace fields and coefficients by

$$(\rho, \mathbf{v}, P, T) \& (\alpha, \mathbf{e}) \longrightarrow (\rho_\varepsilon, \mathbf{v}_\varepsilon, P_\varepsilon, T_\varepsilon) \& (\alpha_\varepsilon, \mathbf{e}_\varepsilon)$$

$$\mu, \eta, \kappa \longrightarrow \varepsilon^2 \mu, \varepsilon^2 \eta, \varepsilon^2 \kappa, \quad (\text{Ansatz from Allard and Atalla, 2009})$$

\Rightarrow non-trivial limit velocity $v_{\varepsilon \rightarrow 0}$, compensating for shrinking pores by reducing friction

Aim of the procedure ("homogenised effect")

Determine the macroscopic behaviour as $\varepsilon \rightarrow 0^+$

Two-scale asymptotic expansion

STEP 1: Assume that each field writes as a power series for the scale ε

$$\rho_\varepsilon(\mathbf{x}, t) = \rho^{(0)}\left(\mathbf{x}, \frac{\mathbf{x}}{\varepsilon}, t\right) + \varepsilon \rho^{(1)}\left(\mathbf{x}, \frac{\mathbf{x}}{\varepsilon}, t\right) + \varepsilon^2 \rho^{(2)}\left(\mathbf{x}, \frac{\mathbf{x}}{\varepsilon}, t\right) + \dots$$

$$\mathbf{v}_\varepsilon(\mathbf{x}, t) = \mathbf{v}^{(0)}\left(\mathbf{x}, \frac{\mathbf{x}}{\varepsilon}, t\right) + \varepsilon \mathbf{v}^{(1)}\left(\mathbf{x}, \frac{\mathbf{x}}{\varepsilon}, t\right) + \varepsilon^2 \mathbf{v}^{(2)}\left(\mathbf{x}, \frac{\mathbf{x}}{\varepsilon}, t\right) + \dots$$

Idem for P_ε , T_ε and $\boldsymbol{\alpha}_\varepsilon$, \mathbf{e}_ε

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Idem for P_ε , T_ε and $\boldsymbol{\alpha}_\varepsilon$, \mathbf{e}_ε

STEP 2: Differential operators and chain rule applied to STEP1

$$\begin{aligned}\partial_t &\equiv \partial_t, & \operatorname{div} &\equiv \operatorname{div}_x + \varepsilon^{-1} \operatorname{div}_y, \\ \nabla &\equiv \nabla_x + \varepsilon^{-1} \nabla_y, & \Delta &\equiv \Delta_x + \varepsilon^{-1} (\operatorname{div}_x \nabla_y + \operatorname{div}_y \nabla_x) + \varepsilon^{-2} \Delta_y.\end{aligned}$$

STEP 3: Inject power series into the governing equations

$$\text{Mass conservation: } \partial_t \sum_{k=0}^{\infty} \varepsilon^k \rho^{(k)} + \rho_0 \sum_{k=0}^{\infty} \varepsilon^k \left(\operatorname{div}_x \mathbf{v}^{(k)} + \varepsilon^{-1} \operatorname{div}_y \mathbf{v}^{(k)} \right) = 0.$$

& similar results for other equations

→ The formal identification of the terms with homogeneous degree ε^k leads to a cascade of equations.

Detailed results for Linearised Navier-Stokes equations

The degrees ε^{-1} and ε^0 are sufficient to describe the homogenised problem.

Mass conservation

$$\varepsilon^{-1} : \quad \operatorname{div}_y \mathbf{v}^{(0)} = 0,$$

$$\varepsilon^0 : \quad \partial_t \rho^{(0)} + \rho_0 \operatorname{div}_x \mathbf{v}^{(0)} + \rho_0 \operatorname{div}_y \mathbf{v}^{(1)} = 0.$$

Momentum conservation

$$\varepsilon^{-1} : \quad \nabla_y P^{(0)} = 0,$$

$$\varepsilon^0 : \quad \rho_0 \partial_t \mathbf{v}^{(0)} = \mu \Delta_y \mathbf{v}^{(0)} + (\zeta + \mu/3) \nabla_y (\operatorname{div}_y \mathbf{v}^{(0)}) \\ - \nabla_x P^{(0)} - \nabla_y P^{(1)}.$$

Thermal conduction

$$\varepsilon^0 : \quad \kappa \Delta_y T^{(0)} - \rho_0 C_p \partial_t T^{(0)} = -\partial_t P^{(0)}.$$

Gas state equation

$$\varepsilon^0 : \quad P^{(0)}/P_0 = \rho^{(0)}/\rho_0 + T^{(0)}/T_0.$$

Boundary conditions :

$$\forall (\mathbf{x}, \mathbf{y}) \in \Omega \times \partial Y_f, \quad \mathbf{v}^{(0)}(\mathbf{x}, \mathbf{y}, t) = \mathbf{v}^{(1)}(\mathbf{x}, \mathbf{y}) = 0,$$

$$\forall (\mathbf{x}, \mathbf{y}) \in \Omega \times \partial Y_f, \quad T^{(0)}(\mathbf{x}, \mathbf{y}, t) = 0.$$

Two-scale Port-Hamiltonian Systems

Two-scale PHS for co-energy variables \mathbf{e}_ε :

$$\mathcal{M} \partial_t \mathbf{e}_\varepsilon = (\mathcal{J}_\varepsilon - \mathcal{R}_\varepsilon) \mathbf{e}_\varepsilon,$$

- constant matrix: \mathcal{M}
- ε -structured energy: $\frac{1}{2} \int_{\Omega_\varepsilon} \sum_{k_1, k_2 \geq 0} \varepsilon^{k_1+k_2} \mathbf{e}^{(k_1)}(\mathbf{x}, \frac{\mathbf{x}}{\varepsilon}, t)^\top \mathcal{M} \mathbf{e}^{(k_2)}(\mathbf{x}, \frac{\mathbf{x}}{\varepsilon}, t) d\Omega$
- ε -structured differential operators $\left\{ \begin{array}{l} \mathcal{J}_\varepsilon = -\mathcal{J}_\varepsilon^* = \sum_{n=-1}^0 \varepsilon^n \mathcal{J}_n \\ \mathcal{R}_\varepsilon = \mathcal{R}_\varepsilon^* = \sum_{n=0}^2 \varepsilon^n \mathcal{R}_n \succeq 0 \end{array} \right.$

with

n	\mathcal{J}_n	\mathcal{R}_n
-1	$\begin{pmatrix} 0 & -\operatorname{div}_y & 0 \\ -\nabla_y & 0 & 0 \\ 0 & 0 & 0 \end{pmatrix}$	$0_{3 \times 3}$
0	$\begin{pmatrix} 0 & -\operatorname{div}_x & 0 \\ -\nabla_x & 0 & 0 \\ 0 & 0 & 0 \end{pmatrix}$	$\begin{pmatrix} 0 & 0 & 0 \\ 0 & -\mathcal{A}_y & 0 \\ 0 & 0 & -\frac{\kappa}{T_0} \Delta_y \end{pmatrix}$
1	$0_{3 \times 3}$	$\begin{pmatrix} 0 & 0 & 0 \\ 0 & -\mathcal{A}_{xy} & 0 \\ 0 & 0 & -\frac{\kappa}{T_0} \Delta_{xy} \end{pmatrix}$
2	$0_{3 \times 3}$	$\begin{pmatrix} 0 & 0 & 0 \\ 0 & -\mathcal{A}_x & 0 \\ 0 & 0 & -\frac{\kappa}{T_0} \Delta_x \end{pmatrix}$

Two-scale PHS and Macroscopic PHS

Two-scale PHS for co-energy variables \mathbf{e}_ε : $\mathcal{M} \partial_t \mathbf{e}_\varepsilon = (\mathcal{J}_\varepsilon - \mathcal{R}_\varepsilon) \mathbf{e}_\varepsilon$,

Details ($\mathcal{S}_n = (\mathcal{J}_n - \mathcal{R}_n)$):

$$\varepsilon^{-1}: \quad 0 = \mathcal{S}_{-1} \mathbf{e}^{(0)},$$

$$\varepsilon^0: \quad \mathcal{M} \partial_t \mathbf{e}^{(0)} = \mathcal{S}_{-1} \mathbf{e}^{(1)} + \mathcal{S}_0 \mathbf{e}^{(0)},$$

$$\varepsilon^1: \quad \mathcal{M} \partial_t \mathbf{e}^{(1)} = \mathcal{S}_{-1} \mathbf{e}^{(2)} + \mathcal{S}_0 \mathbf{e}^{(1)} + \mathcal{S}_1 \mathbf{e}^{(0)},$$

$$\varepsilon^k, k \geq 2: \quad \mathcal{M} \partial_t \mathbf{e}^{(k)} = \mathcal{S}_{-1} \mathbf{e}^{(k+1)} + \mathcal{S}_0 \mathbf{e}^{(k)} + \mathcal{S}_1 \mathbf{e}^{(k-1)} + \mathcal{S}_2 \mathbf{e}^{(k-2)}.$$

The homogenised problem (ε^{-1} & ε^0) defines a **Macroscopic PHS**

$$\begin{pmatrix} \mathcal{M} \partial_t \mathbf{e}^{(0)} \\ 0 \end{pmatrix} = \underbrace{\begin{pmatrix} \mathcal{S}_0 & \mathcal{S}_{-1} \\ \mathcal{S}_{-1} & 0 \end{pmatrix}}_{\text{negative op.}} \begin{pmatrix} \mathbf{e}^{(0)} \\ \mathbf{e}^{(1)} \end{pmatrix},$$

Remark: $\mathbf{e}^{(1)}$ serves as the Lagrange multiplier.

Two-scale PHS and Macroscopic PHS

Two-scale PHS for energy variables α_ε

$$\underbrace{\partial_t \alpha_\varepsilon}_{\mathcal{M} \partial_t \mathbf{e}_\varepsilon} = \underbrace{(\mathcal{J}_\varepsilon - \mathcal{R}_\varepsilon)}_{\mathcal{S}_\varepsilon} \underbrace{\delta_{\alpha_\varepsilon} H_\varepsilon(\alpha_\varepsilon)}_{\mathcal{M}^{-1} \alpha_\varepsilon = \mathbf{e}_\varepsilon},$$

with $\alpha_\varepsilon := \mathcal{M} \mathbf{e}_\varepsilon$

$$\text{and } H_\varepsilon(\alpha_\varepsilon) := \frac{1}{2} \langle \mathcal{M}^{-1} \alpha_\varepsilon, \alpha_\varepsilon \rangle_{\Omega_\varepsilon},$$

The homogenised problem (ε^{-1} & ε^0) defines a Macroscopic PHS

$$\begin{pmatrix} \partial_t \alpha^{(0)} \\ 0 \end{pmatrix} = \begin{pmatrix} \mathcal{S}_0 & \mathcal{S}_{-1} \\ \mathcal{S}_{-1} & 0 \end{pmatrix} \begin{pmatrix} \mathcal{M}^{-1} \alpha^{(0)} \\ \mathcal{M}^{-1} \alpha^{(1)} \end{pmatrix}.$$

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Homogenised fields

Denote the fields with a hat symbol in the Laplace domain

$s \in \overline{\mathbb{C}}_0^+ = \{s \in \mathbb{C} \mid \operatorname{Re}(s) \geq 0\}$ (or $s = i\omega \in i\mathbb{R}$ for the Fourier domain)

Fields $\hat{P}^{(0)}$, $\hat{T}^{(0)}$ and $\hat{v}^{(0)}$

$$\hat{T}^{(0)}(\mathbf{x}, \mathbf{y}) = \hat{P}^{(0)}(\mathbf{x}) \varphi(\mathbf{y}),$$

$$\hat{v}^{(0)}(\mathbf{x}, \mathbf{y}) = \sum_{i=1}^d \mathbf{w}_i(\mathbf{y}) \frac{\partial \hat{P}^{(0)}}{\partial x_i}(\mathbf{x}),$$

where φ and (\mathbf{w}_i, q_i) are solutions to boundary problems in the unit cell :

$$\begin{cases} \kappa \Delta \varphi(\mathbf{y}) - s \rho_0 C_p \varphi(\mathbf{y}) = -s & \text{on } Y_f, \\ \varphi|_{\partial Y_f} = 0, \end{cases} \quad (5)$$

$$\begin{cases} \mu \Delta \mathbf{w}_i(\mathbf{y}) - \nabla q_i(\mathbf{y}) - s \rho_0 \mathbf{w}_i(\mathbf{y}) = \mathbf{E}_i & \text{on } Y_f \\ \operatorname{div} \mathbf{w}_i(\mathbf{y}) = 0 & \text{on } Y_f \\ \mathbf{w}_i|_{\partial Y_f} = 0. \end{cases} \quad (6)$$

\mathbf{E}_i : unit vector oriented in direction i

Effective fluid and loss coefficients in acoustic pipes

Define the permeability operators

$$\hat{\mathbf{A}}\mathbf{E}_i = \langle \mathbf{w}_i \rangle \text{ and } \hat{B} = \langle \varphi \rangle$$

with the average operator $\langle \cdot \rangle = \int_{Y_f} \cdot \, d\mathbf{y} / \int_{Y_f} 1 \, d\mathbf{y}$.

Result

(see [Thi23, chap. 6])

Fields $\hat{P}^{(0)}$, $\hat{T}^{(0)}$ and $\hat{\mathbf{v}}^{(0)}$ correspond to an effective fluid characterised by

$$\rho_e(s) = -\frac{1}{s} \hat{\mathbf{A}}^{-1}(s), \quad (\text{density})$$

$$K_e(s) = \frac{1}{1/P_0 - \hat{B}(s)/T_0}, \quad (\text{compressibility})$$

Loss coefficients (analog to those involved in acoustic pipes with boundary layers) that represent the macroscopic dissipation inside the porous wall are

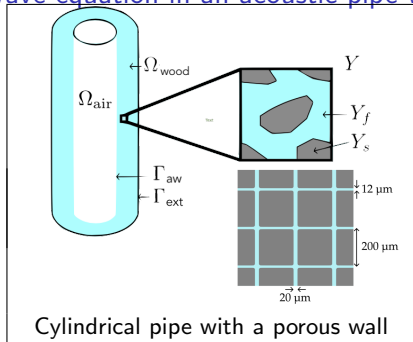
$$K_t(s) = 1 - \rho_0 C_p \hat{B}(s), \quad (\text{thermal})$$

$$K_v(s) = I_d + \rho_0 s \hat{\mathbf{A}}(s), \quad (\text{viscous})$$

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Wave equation in an acoustic pipe with a porous wall



$$\begin{aligned}\Omega_{\text{air}} &= \{ \mathbf{x} \mid r < R \}, \\ \Omega_{\text{wood}} &= \{ \mathbf{x} \mid R < r < R_{\text{ext}} \}, \\ \Gamma_{\text{ab}} &= \{ \mathbf{x} \mid r = R \}, \\ \Gamma_{\text{ext}} &= \{ \mathbf{x} \mid r = R_{\text{ext}} \}.\end{aligned}$$

Wave propagation in $\Omega_{\text{air}} \cup \Omega_{\text{wood}}$ is governed by

$$\tilde{\rho}_e(s, \mathbf{x}) s \tilde{\mathbf{v}}(s, \mathbf{x}) = -\nabla \hat{P}^{(0)}(s, \mathbf{x}) \quad \& \quad s \hat{P}^{(0)}(s, \mathbf{x}) = -\tilde{K}_e(s, \mathbf{x}) \operatorname{div} \tilde{\mathbf{v}}(s, \mathbf{x})$$

$$\text{with } \begin{cases} \tilde{\mathbf{v}} = \hat{\mathbf{v}}, & \tilde{K}_e = \gamma P_0, & \tilde{\rho}_e = \rho_0 & \text{in } \Omega_{\text{air}}, \\ \tilde{\mathbf{v}}(s, \mathbf{x}) = \phi \langle \hat{\mathbf{v}} \rangle(s, \mathbf{x}), & \tilde{K}_e(s, \mathbf{x}) = \frac{K_e(s)}{\phi}, & \tilde{\rho}_e(s, \mathbf{x}) = \frac{\rho_e(s)}{\phi} & \text{in } \Omega_{\text{bois}} \end{cases}$$

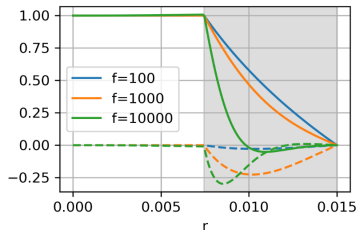
and porosity coefficient $\phi := |Y_f|/|Y|$.

Helmholtz equation (eliminating $\tilde{\mathbf{v}}$)

$$s^2 \tilde{K}_e^{-1}(s, \mathbf{x}) \hat{P}^{(0)}(s, \mathbf{x}) - \operatorname{div}(\tilde{\rho}_e^{-1}(s, \mathbf{x}) \nabla \hat{P}^{(0)}(s, \mathbf{x})) = 0 \text{ in } \Omega \quad \& \quad \hat{P}^{(0)} = 0 \text{ on } \Gamma_{\text{ext}}$$

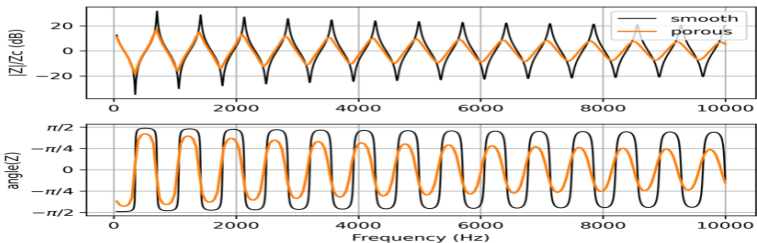
Numerical results

1. Pipe parameters: $R = 7.45$ mm, $R_{\text{ext}} = 15.0$ mm, porosity $\phi = 0.156$
2. Solve the cell problem (high-order finite element code *Montjoie*) (see [Dur21])
3. Derive the dispersion relationship of the first pressure mode (closest to the planar mode).



Radial profile of the first pressure mode:
real (-) and imaginary(--) parts

Normalised input impedance for the length $L = 240.5$ mm



(black) perfect wall (Zwikker-Kosten model);
(orange) porous wall (academic geometry with exaggerated porosity)

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Conclusion

Interest of combining the homogenisation method and the port-Hamiltonian formulation

- enables the passivity of microscopic phenomena to be characterised at several scales
- produces passive macroscopic descriptions
- interpretation of microscopic / two-scale / macroscopic energies and operators

Perspectives

- Convergence analysis of the solutions
- Examine this approach on nonlinear systems.

Thank you for your attention

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