

Analysis of one-dimensional structure using Lie groups approach

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Joint work with Loïc Le Marrec and Jean Lerbet

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Mathematical tools

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- $\mathbb{D}(\mathcal{E})$ group of Euclidean displacements and $\mathfrak{D}(\mathcal{E})$ the associated Lie algebra.
- $\mathfrak{D}(\mathcal{E}) = \{X : \mathcal{E} \longrightarrow E, \exists \omega_X \in E / X(b) = X(a) + \omega_X \times \vec{ab} \quad \forall a, b \in \mathcal{E}\}$
where \times is the standard cross product in E .

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- The Lie bracket is defined in $\mathfrak{D}(\mathcal{E})$ by

$$\forall a \in \mathcal{E} \quad [X, Y](a) = \omega_X \times Y(a) - \omega_Y \times X(a)$$

Klein form

The Klein form $[[\cdot|\cdot]]$ on $\mathfrak{D}(\mathcal{E})$ is the map defined by:

$$\begin{aligned} [[\cdot|\cdot]] : \mathfrak{D}(\mathcal{E}) \times \mathfrak{D}(\mathcal{E}) &\longrightarrow \mathbb{R} \\ (X, Y) &\longrightarrow [[X|Y]] = \omega_X \cdot Y + \omega_Y \cdot X \end{aligned}$$

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- $[[\cdot|\cdot]]$ is a bilinear, symmetric and nondegenerate form on $\mathcal{D}(\mathcal{E})$. Its signature is $(3, 3)$.

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- $[[\cdot|\cdot]]$ is a bilinear, symmetric and nondegenerate form on $\mathcal{D}(\mathcal{E})$. Its signature is $(3, 3)$.
- $[[\cdot|\cdot]]$ is invariant by applying the adjoint map.
- $[[\cdot|\cdot]]$ plays a crucial role in energy-related aspects.

Kinematics

Internal motion

$$\begin{aligned}g : [0, L_0] \times \mathbb{R}^+ &\longrightarrow \mathbb{R} \times \mathbb{R}^+ \\(S_0, t) &\longrightarrow (S(t), t)\end{aligned}$$

$S_0 \in [0, L_0]$ initial configuration and $S \in [0, L(t)]$ deformed configuration.

Kinematics

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Configuration

$$p(S, t) = D(S_0, t) \bullet p_0(S_0),$$

$D(S_0, t)$ displacement.

$p_0(S_0)$ reference configuration.

• free and transitive left-action of \mathbb{D} on the configuration space.

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Strain

$$\begin{aligned} \mathbf{e}_0 : [0, L_0] \times \mathbb{R}^+ &\longrightarrow \mathfrak{D}(\mathcal{E}) \\ (S_0, t) &\longrightarrow \mathbf{e}_0(S_0, t) = \vartheta\left(\frac{\partial D(S_0, t)}{\partial S_0}\right) = \mathbf{D}^{-1}(S_0, t) \circ \frac{\partial D(S_0, t)}{\partial S_0} \end{aligned}$$

Where \mathbf{D} linear part of D .

Dynamics

Deformed configuration (S, t)

$$\rho \mathcal{T} + \frac{\partial \Theta}{\partial S} = \frac{\partial}{\partial t} (\rho H(V))$$

- $\rho \mathcal{T}$ force distribution.
- V velocity
- Θ Internal actions
- ρH inertia

Dynamics

Deformed configuration (S, t)

$$\rho \mathcal{T} + \frac{\partial \Theta}{\partial S} = \frac{\partial}{\partial t} (\rho H(V))$$

- $\rho(S, t)dS = \rho(S_0)dS_0$
- $U(S, t) = Ad(D(S_0, t))U(S_0, t) \quad \forall U \in \mathcal{D}(\mathcal{E})$
- $[AdDX, AdDY] = AdD[X, Y]$

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- $[AdDX, AdDY] = AdD[X, Y]$

Initial configuration (S_0, t)

$$\rho_0 \mathcal{T}_0 + \frac{\partial \Theta_0}{\partial S_0} + [\mathbf{e}_0, \Theta_0] = \rho_0 \left(H_0 \left(\frac{\partial V_0}{\partial t} \right) + [V_0, H_0(V_0)] \right)$$

Kinematics

Displacement decomposition

$$\mathbb{D}(\mathcal{E}) = \mathbb{T}(\mathcal{E}) \times_C \mathbb{R}(\mathcal{E})_C$$

- \times_C semi-direct product
- C center of mass depends on S_0
- $\mathbb{T}(\mathcal{E})$ translation group
- $\mathbb{R}(\mathcal{E})$ rotation group

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Displacement decomposition

$$\mathcal{D}(\mathcal{E}) = \mathbb{T}(\mathcal{E}) \times_C \mathbb{R}(\mathcal{E})_C$$

Lie algebra

$$\mathfrak{D}(\mathcal{E}) = \mathfrak{T} \oplus \mathfrak{R}_C$$

- \times_C semi-direct product
- C center of mass depends on S_0
- $\mathbb{T}(\mathcal{E})$ translation group
- $\mathbb{R}(\mathcal{E})$ rotation group

- $\mathfrak{T} = \{X \in \mathfrak{D}(\mathcal{E}) | \omega_X = 0\}$
- $\mathfrak{R}_C = \{X \in \mathfrak{D}(\mathcal{E}) | X(C) = 0\}$

Kinematics

Displacement decomposition

$$\mathcal{D}(\mathcal{E}) = \mathcal{T}(\mathcal{E}) \times_C \mathcal{R}(\mathcal{E})_C$$

Lie algebra

$$\mathcal{D}(\mathcal{E}) = \mathcal{I} \oplus \mathfrak{R}_C$$

Strain

$$\mathbf{e}_0(S_0, t) = \underbrace{\varepsilon_0(S_0, t)}_{\in \mathcal{I}} + \underbrace{\kappa_0(S_0, t)}_{\in \mathfrak{R}_C}$$

- \times_C semi-direct product
- C center of mass depends on S_0
- $\mathcal{T}(\mathcal{E})$ translation group
- $\mathcal{R}(\mathcal{E})$ rotation group
- $\mathcal{I} = \{X \in \mathcal{D}(\mathcal{E}) | \omega_X = 0\}$
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Statics

Forces and moments

$$\Theta_0(S_0, t) = \underbrace{\mathbf{M}_0(S_0, t)}_{\in \mathfrak{I}} + \underbrace{\mathbf{F}_0(S_0, t)}_{\in \mathfrak{R}_c}$$

Statics

Forces and moments

$$\Theta_0(S_0, t) = \underbrace{\mathbf{M}_0(S_0, t)}_{\in \mathfrak{I}} + \underbrace{\mathbf{F}_0(S_0, t)}_{\in \mathfrak{R}_C}$$

Statics-level one

$$\rho_0 \mathcal{T}_0 + \frac{\partial \Theta_0}{\partial S_0} + [\mathbf{e}_0, \Theta_0] = 0$$

$$\Downarrow \rho_0 \mathcal{T}_0 = \underbrace{\mathbf{m}}_{\in \mathfrak{I}} + \underbrace{\mathbf{q}}_{\in \mathfrak{R}_C}$$

Statics-level two

$$\begin{cases} \frac{d_C(S_0) \mathbf{F}_0}{dS_0} + [\boldsymbol{\kappa}_0, \mathbf{F}_0] + \mathbf{q} = 0 \\ \frac{d_C(S_0) \mathbf{M}_0}{dS_0} + [\boldsymbol{\varepsilon}_0, \mathbf{F}_0] + [\boldsymbol{\kappa}_0, \mathbf{M}_0] + \mathbf{m} = 0 \end{cases}$$

Statics

$(\mathbf{e}_1, \mathbf{e}_2, \mathbf{e}_3)$ basis in \mathcal{T}

(ξ_1, ξ_2, ξ_3) basis in \mathcal{R}_C

Statics

$(\mathbf{e}_1, \mathbf{e}_2, \mathbf{e}_3)$ basis in \mathfrak{T}

(ξ_1, ξ_2, ξ_3) basis in \mathfrak{R}_C

Deformation | Force

$$\boldsymbol{\varepsilon}_0 = \sum_{i=1}^3 \varepsilon_i \mathbf{e}_i \quad \Bigg| \quad \mathbf{F}_0 = \sum_{i=1}^3 F_i \xi_i$$

Curvature | Moment

$$\boldsymbol{\kappa}_0 = \sum_{i=1}^3 \kappa_i \xi_i \quad \Bigg| \quad \mathbf{M}_0 = \sum_{i=1}^3 M_i \mathbf{e}_i$$

Statics

$(\mathbf{e}_1, \mathbf{e}_2, \mathbf{e}_3)$ basis in \mathfrak{T}

(ξ_1, ξ_2, ξ_3) basis in \mathfrak{R}_C

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Curvature | Moment

$$\boldsymbol{\kappa}_0 = \sum_{i=1}^3 \kappa_i \xi_i \quad \Bigg| \quad \mathbf{M}_0 = \sum_{i=1}^3 M_i \mathbf{e}_i$$

$$\frac{d_C(s_0) \mathbf{F}_0}{ds_0} + [\boldsymbol{\kappa}_0, \mathbf{F}_0] + \mathbf{q} = 0$$

$$\frac{d_C(s_0) \mathbf{M}_0}{ds_0} + [\boldsymbol{\varepsilon}_0, \mathbf{F}_0] + [\boldsymbol{\kappa}_0, \mathbf{M}_0] + \mathbf{m} = 0$$



Straight beam



Straight beam

$$\begin{aligned} F_1' - F_2 \kappa_3 + F_3 \kappa_2 + q_1 &= 0 \\ F_2' + F_1 \kappa_3 - F_3 \kappa_1 + q_2 &= 0 \\ F_3' - F_1 \kappa_2 + F_2 \kappa_1 + q_3 &= 0 \end{aligned}$$

$$\begin{aligned} M_1' - M_2 \kappa_3 + M_3 \kappa_2 - F_2(\varepsilon_3 + 1) + F_3 \varepsilon_2 + m_1 &= 0 \\ M_2' - M_3 \kappa_1 + M_1 \kappa_3 - F_3 \varepsilon_1 + F_1(\varepsilon_3 + 1) + m_2 &= 0 \\ M_3' - M_1 \kappa_2 + M_2 \kappa_1 - F_1 \varepsilon_2 + F_2 \varepsilon_1 + m_3 &= 0 \end{aligned}$$

Constitutive laws

Linear constitutive law | Level one

$$\Theta_0 = \mathfrak{E}(\mathbf{e}_0), \quad \mathfrak{E} : \mathfrak{D}(\mathcal{E}) \rightarrow \mathfrak{D}(\mathcal{E}), \quad \text{linear}$$

Elastic energy | Level one

$$E_{el} = \frac{1}{2} [[\mathfrak{E}(\mathbf{e}_0) | \mathbf{e}_0]]$$

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Assume that \mathfrak{E} is symmetric with respect to the Klein form:

$$[[\mathfrak{E}(u)|v]] = [[u|\mathfrak{E}(v)]], \quad \forall u, v \in \mathfrak{D}.$$

Elastic energy|Level two

$$E_{el} = \frac{1}{2} \left([[\mathfrak{E}(\varepsilon_0)|\varepsilon_0]] + 2[[\mathfrak{E}(\varepsilon_0)|\kappa_0]] + [[\mathfrak{E}(\kappa_0)|\kappa_0]] \right)$$

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Elastic energy|Level two

$$E_{el} = \frac{1}{2} \left([[\mathfrak{E}(\varepsilon_0) | \varepsilon_0]] + 2[[\mathfrak{E}(\varepsilon_0) | \kappa_0]] + [[\mathfrak{E}(\kappa_0) | \kappa_0]] \right)$$

Using a normalized basis $\mathcal{B} = (\mathbf{e}_1, \mathbf{e}_2, \mathbf{e}_3, \xi_1, \xi_2, \xi_3)$ that diagonalizes the symmetrical operator

Elastic energy|Level three

$$E_{el} = \frac{1}{2} \left(E I_1 \varepsilon_1^2 + E I_2 \varepsilon_2^2 + G I_3 \varepsilon_3^2 + E I_1 \kappa_1^2 + E I_2 \kappa_2^2 + G I_3 \kappa_3^2 \right)$$

Problem statement

- Beam at an equilibrium and subjected to linear perturbation.
- Level one equation.
- $D(S_0, t) = \exp X(S_0, t) \circ D_0(S_0)$ where
 - $D_0(S_0)$ solution of the equilibrium equation
 - $X(S_0, t) \in \mathcal{D}(\mathcal{E})$ the unknown kinematic of the vibration problem.

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equilibrium

$$\rho_0 \mathcal{T}_0(S_0) + \frac{\partial \Theta_{e,0}}{\partial S_0}(S_0) + [\mathbf{e}_{e,0}(S_0), \Theta_{e,0}(S_0)] = 0$$

- $\mathbf{e}_{e,0}(S_0)$ Lagrangian deformation at equilibrium.
- $\Theta_{e,0}(S_0)$ Lagrangian internal actions at equilibrium
- $\mathcal{T}_0(S_0)$ Lagrangian external actions at the equilibrium

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- $\mathcal{T}_0(S_0)$ Lagrangian external actions at the equilibrium

Note that

No additional perturbation is added to $\mathcal{T}_0(S_0)$.

Linear approximation calculation

$$\rho_0 \mathcal{T}_0 + \frac{\partial \Theta_0}{\partial S_0} + [\mathbf{e}_0, \Theta_0] = \rho_0 \left(H_0 \left(\frac{\partial V_0}{\partial t} \right) + [V_0, H_0(V_0)] \right)$$

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$$\begin{aligned} \mathbf{e}_0(S_0, t) &\approx \text{Ad } D_0^{-1}(S_0) \frac{\partial X(S_0, t)}{\partial S_0} + \mathbf{e}_{e,0}(S_0) \\ \Theta_0(S_0, t) &= \Theta_{e,0}(S_0) + \chi_0(S_0, t) \end{aligned}$$

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$$\begin{aligned} V_0(S_0, t) &\approx \text{Ad } D_0^{-1}(S_0) \frac{\partial X(S_0, t)}{\partial t} \\ \frac{\partial V_0(S_0, t)}{\partial t} &\approx \text{Ad } D_0^{-1}(S_0) \frac{\partial^2 X(S_0, t)}{\partial t^2} \end{aligned}$$

Linear approximation calculation

$$\rho_0 \mathcal{T}_0 + \frac{\partial \Theta_0}{\partial S_0} + [\mathbf{e}_0, \Theta_0] = \rho_0 \left(H_0 \left(\frac{\partial V_0}{\partial t} \right) + [V_0, H_0(V_0)] \right)$$

$$\mathbf{e}_0(S_0, t) \approx \text{Ad } D_0^{-1}(S_0) \frac{\partial X(S_0, t)}{\partial S_0} + \mathbf{e}_{e,0}(S_0)$$

$$\Theta_0(S_0, t) = \Theta_{e,0}(S_0) + \chi_0(S_0, t)$$

$$V_0(S_0, t) \approx \text{Ad } D_0^{-1}(S_0) \frac{\partial X(S_0, t)}{\partial t}$$

$$\frac{\partial V_0(S_0, t)}{\partial t} \approx \text{Ad } D_0^{-1}(S_0) \frac{\partial^2 X(S_0, t)}{\partial t^2}$$

Perturbed equation

$$\frac{\partial \chi_0}{\partial S_0} + [\text{Ad } D_0^{-1}(S_0) \left(\frac{\partial X}{\partial S_0} \right), \Theta_{e,0}(S_0)] + [\mathbf{e}_{e,0}(S_0), \chi_0] = \rho_0 \left(H_0 \left(\text{Ad } D_0^{-1}(S_0) \frac{\partial^2 X}{\partial t^2} \right) \right)$$

Dynamic equation

Linear elastic constitutive law

$$\chi_0(S_0, t) = \mathfrak{E}_0 \left(\text{Ad } D_0^{-1}(S_0) \frac{\partial X(S_0, t)}{\partial S_0} \right)$$

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- Applying the left action of $\text{Ad } D_0(S_0)$ on perturbed equation.
- For all $Y \in \mathfrak{D}(\mathcal{E})$, we let

$$\tilde{\mathbf{e}}_{e,0}(S_0) = \text{Ad } D_0(S_0)(\mathbf{e}_{e,0}(S_0))$$

$$\tilde{H}_0(Y) = \text{Ad } D_0(S_0) \circ H_0 \circ \text{Ad } D_0^{-1}(S_0)(Y)$$

$$\tilde{\Theta}_{e,0}(S_0) = \text{Ad } D_0(S_0)(\Theta_{e,0}(S_0))$$

$$\tilde{\mathfrak{E}}_0(Y) = \text{Ad } D_0(S_0) \circ \mathfrak{E}_0 \circ \text{Ad } D_0^{-1}(S_0)(Y)$$

$$\tilde{R}_0(Y) = [Y, \tilde{\Theta}_{e,0}(S_0)] + [\tilde{\mathbf{e}}_{e,0}(S_0), \tilde{\mathfrak{E}}_0(Y)] + \tilde{\mathfrak{E}}_0([Y, \tilde{\mathbf{e}}_{e,0}(S_0)])$$

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Perturbed equation

$$\tilde{\mathfrak{E}}_0 \left(\frac{\partial^2 X(S_0, t)}{\partial S_0^2} \right) + \tilde{R}_0 \left(\frac{\partial X(S_0, t)}{\partial S_0} \right) = \rho_0 \tilde{H}_0 \left(\frac{\partial^2 X(S_0, t)}{\partial t^2} \right)$$

Variational formulation

- Boundary conditions should be given to define a well-posed problem.
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- Dynamic properties imply that \tilde{H}_0 is a symmetric operator with respect to the Klein form.

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$$\tilde{\mathcal{E}}_0\left(\frac{\partial^2 X(S_0, t)}{\partial S_0^2}\right) + \tilde{R}_0\left(\frac{\partial X(S_0, t)}{\partial S_0}\right)$$

Replace $X(S_0, t)$ by $U(S_0)$

$$\tilde{\mathcal{E}}_0(U''(S_0)) + \tilde{R}_0(U'(S_0))$$

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Replace $X(S_0, t)$ by $U(S_0)$

$$\tilde{\mathcal{E}}_0(U''(S_0)) + \tilde{R}_0(U'(S_0))$$

Integrate along the beam

$$\Psi_0(U, V) = \int_0^{L_0} [[\tilde{\mathcal{E}}_0(U''(S_0)) + \tilde{R}_0(U'(S_0)) | V(S_0)]] dS_0$$

Variational formulation

Integration by parts

$$\begin{aligned}
 \Psi_0(U, V) = & - \underbrace{\int_0^{L_0} [(\tilde{\mathfrak{E}}_0(U')) | V'] dS_0}_{\text{Symmetrical operator}} + \\
 & \underbrace{\int_0^{L_0} [([U', \tilde{\mathfrak{E}}_0(\tilde{\mathbf{e}}_{e,0})]) | V] dS_0}_{\mathfrak{B}(U, V)} + \underbrace{[[(\tilde{\mathfrak{E}}_0(U')) | V]]_0^{L_0}}_{\text{Boundary condition}}
 \end{aligned}$$

- Symmetry of $\mathfrak{B}(U, V) \implies$ symmetry of $\Psi_0(U, V)$.

Variational formulation

Integration by parts

$$\Psi_0(U, V) = - \underbrace{\int_0^{L_0} [(\tilde{\mathfrak{E}}_0(U')) | V'] dS_0}_{\text{Symmetrical operator}} + \underbrace{\int_0^{L_0} [([U', \tilde{\mathfrak{E}}_0(\tilde{\mathbf{e}}_{e,0})]) | V] dS_0}_{\mathfrak{B}(U, V)} + \underbrace{[[(\tilde{\mathfrak{E}}_0(U')) | V]]_0^{L_0}}_{\text{Boundary condition}}$$

- Symmetry of $\mathfrak{B}(U, V) \implies$ symmetry of $\Psi_0(U, V)$.

$$\mathfrak{B}(U, V) - \mathfrak{B}(V, U) = - \int_0^{L_0} [([U, V] | \tilde{\mathfrak{E}}_0(\tilde{\mathbf{e}}_{e,0})')] dS_0 + [([([U, V] | \tilde{\mathfrak{E}}_0(\tilde{\mathbf{e}}_{e,0})))]_0^{L_0}$$

Variational formulation

Integration by parts

$$\Psi_0(U, V) = - \underbrace{\int_0^{L_0} [[(\tilde{\mathfrak{E}}_0(U')) \mid V']] dS_0}_{\text{Symmetrical operator}} + \underbrace{\int_0^{L_0} [[[U', \tilde{\mathfrak{E}}_0(\tilde{\mathbf{e}}_{e,0})] \mid V]] dS_0}_{\mathfrak{B}(U, V)} + \underbrace{[[[(\tilde{\mathfrak{E}}_0(U')) \mid V]]]_0^{L_0}}_{\text{Boundary condition}}$$

- Symmetry of $\mathfrak{B}(U, V) \implies$ symmetry of $\Psi_0(U, V)$.

$$\mathfrak{B}(U, V) - \mathfrak{B}(V, U) = - \int_0^{L_0} [[[U, V] \mid \tilde{\mathfrak{E}}_0(\tilde{\mathbf{e}}_{e,0})']] dS_0 + [[[[U, V] \mid \tilde{\mathfrak{E}}_0(\tilde{\mathbf{e}}_{e,0})]]]_0^{L_0}$$

Examples : $\mathfrak{B}(U, V)$ is symmetric if

$$\tilde{\mathfrak{E}}_0(\tilde{\mathbf{e}}_{e,0}) = 0$$

Clamped-Clamped beam and a uniform $\tilde{\mathfrak{E}}_0(\tilde{\mathbf{e}}_{e,0})$

Conclusion and perspectives

- Exploration of one-dimensional structures through the application of Lie group structures.
- Three levels of equation setting
 - First level \implies a single equation within the Lie algebra of the displacement group $\mathbb{D}(\mathcal{E})$.
 - Second level \implies semi-direct product $\mathbb{D}(\mathcal{E}) = \mathbb{T}(\mathcal{E}) \times_{\mathcal{C}} \mathbb{R}(\mathcal{E})_{\mathcal{C}}$.
 - The third level \implies introduction of an appropriate basis.
- Perturbation of linear dynamics around an arbitrary equilibrium position :strong and weak formulations.
- Perspectives :
 - Boundary conditions investigations.
 - Deformed cross sections.